

## Solution of Block Diagram

Manipulate block diagram to solve for  $\hat{v}(s)$ . Result is

$$\hat{v} = \hat{v}_{ref} \frac{G_c G_{vd} / V_M}{1 + H G_c G_{vd} / V_M} + \hat{v}_g \frac{G_{vg}}{1 + H G_c G_{vd} / V_M} - \hat{i}_{load} \frac{Z_{out}}{1 + H G_c G_{vd} / V_M}$$

which is of the form

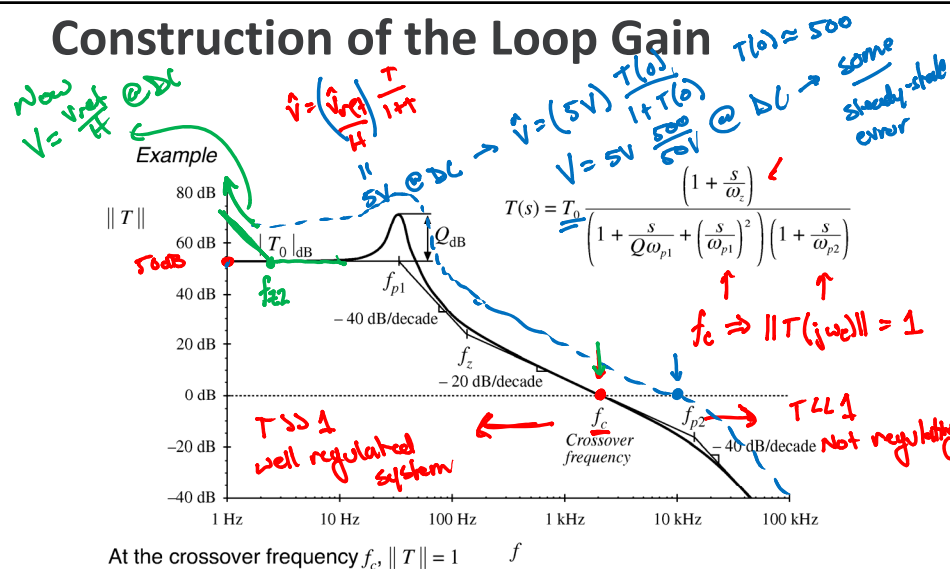
$$\hat{v} = \hat{v}_{ref} \frac{1}{H} \frac{T}{1+T} + \hat{v}_g \frac{G_{vg}}{1+T} - \hat{i}_{load} \frac{Z_{out}}{1+T}$$

if  $T \rightarrow \infty$   
 $\hat{v} \approx \hat{v}_{ref} \frac{1}{H}$

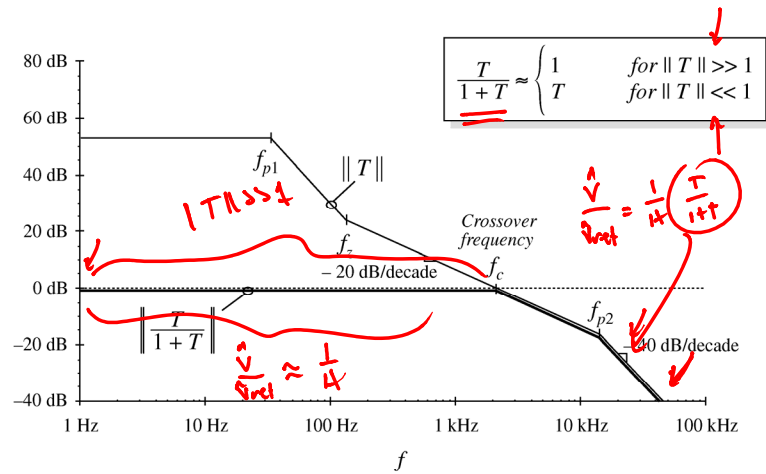
with  $T(s) = H(s) G_c(s) G_{vd}(s) / V_M = \text{"loop gain"}$

Loop gain  $T(s)$  = products of the gains around the negative feedback loop.

## Construction of the Loop Gain



## Effect of Loop Gain

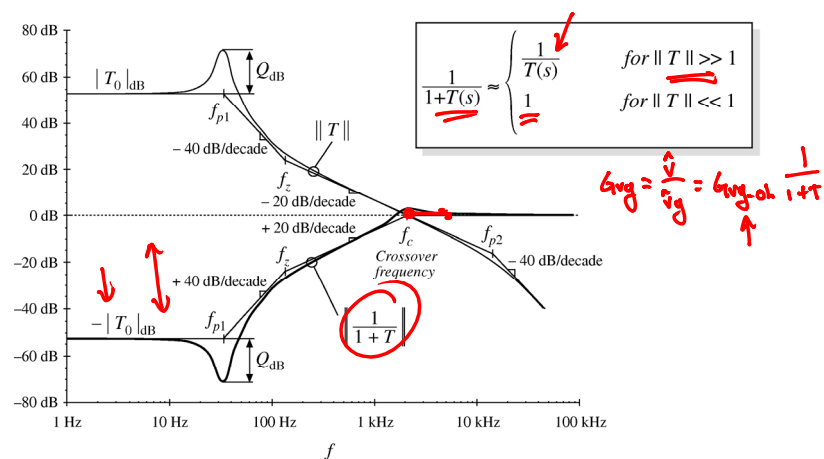


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## Rejection of Disturbances



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## Open-Loop and Closed-Loop

Original transfer functions, before introduction of feedback ("open-loop transfer functions"):

$$G_{vd}(s) \quad G_{vg}(s) \quad Z_{out}(s)$$

Upon introduction of feedback, these transfer functions become ("closed-loop transfer functions"):

$$\rightarrow \frac{1}{H(s)} \frac{T(s)}{1+T(s)} \quad \frac{G_{vg}(s)}{1+T(s)} \quad \frac{Z_{out}(s)}{1+T(s)}$$

The loop gain:

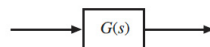
$$T(s)$$

## Effect of Feedback

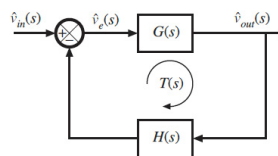
Feedback moves the poles of the system transfer functions

- Good news: we can use feedback to alter the poles and improve the frequency response
- Bad news: if you're not careful, feedback can move the poles into the right half of the complex s-plane (poles have positive real parts), leading to an unstable system

Open loop



Closed loop



## Determining Stability From $T(s)$

- Nyquist stability theorem: general result.
- A special case of the Nyquist stability theorem: the phase margin test  
 Allows determination of closed-loop stability (i.e., whether  $1/(1+T(s))$  contains RHP poles) directly from the magnitude and phase of  $T(s)$ .  
 A good design tool: yields insight into how  $T(s)$  should be shaped, to obtain good performance in transfer functions containing  $1/(1+T(s))$  terms.

Phase margin test is valid if:

- (1)  $f_c \Rightarrow \|T(j2\pi f_c)\| = 1$  is uniquely defined.
- (2)  $T(s)$  has no RHP poles

### 9.4.1 – The Phase Margin Test

'Phase Margin' =  $\varphi_m = 180^\circ + \angle T(j\omega_c)$

A test on  $T(s)$ , to determine whether  $1/(1+T(s))$  contains RHP poles.

The crossover frequency  $f_c$  is defined as the frequency where

$$\|T(j2\pi f_c)\| = 1 \Rightarrow 0\text{dB}$$

The phase margin  $\varphi_m$  is determined from the phase of  $T(s)$  at  $f_c$ , as follows:

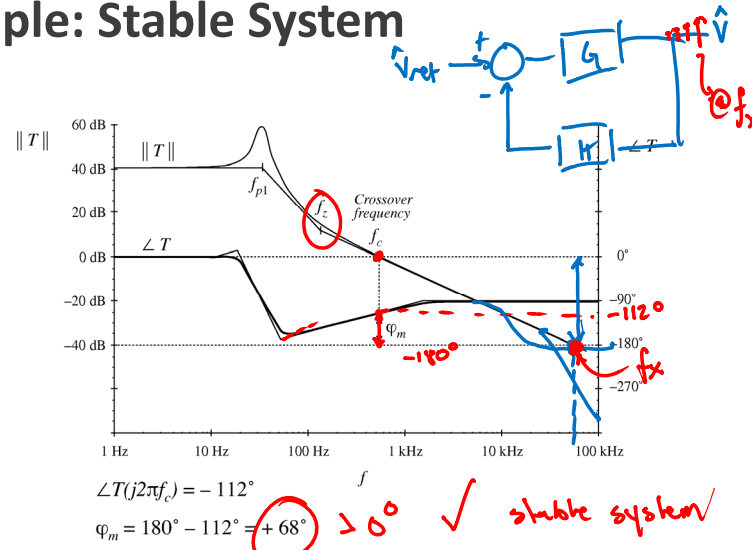
$$\varphi_m = 180^\circ + \angle T(j2\pi f_c)$$

system stable when  
 $\varphi_m > 0^\circ$

If there is exactly one crossover frequency, and if  $T(s)$  contains no RHP poles, then

the quantities  $T(s)/(1+T(s))$  and  $1/(1+T(s))$  contain no RHP poles whenever the phase margin  $\varphi_m$  is positive.

## Example: Stable System

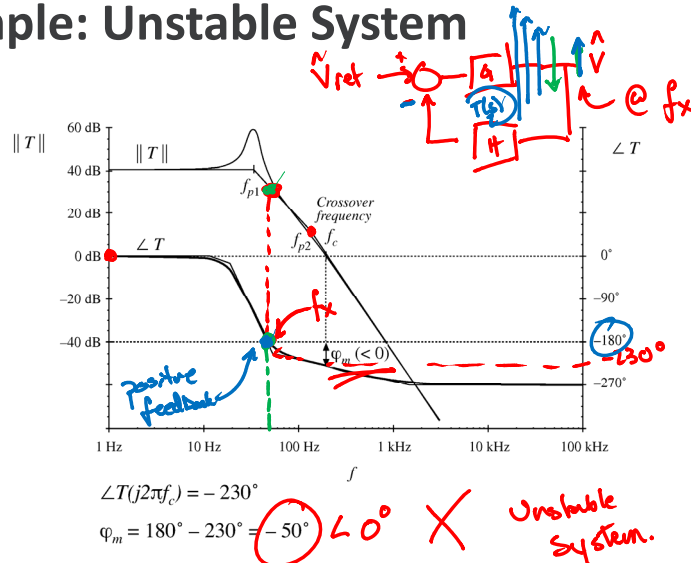


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## Example: Unstable System



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## Selecting Phase Margin

*stable for  $\varphi_m > 0^\circ$*

How much phase margin is required?

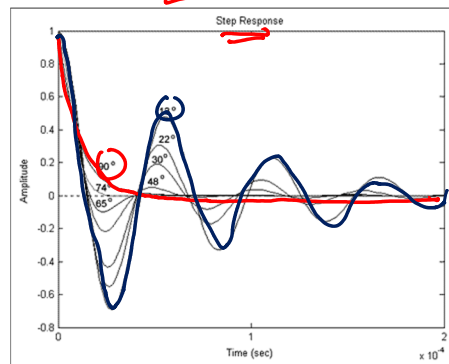
A small positive phase margin leads to a stable closed-loop system having complex poles near the crossover frequency with high  $Q$ . The transient response exhibits overshoot and ringing.

Increasing the phase margin reduces the  $Q$ . Obtaining real poles, with no overshoot and ringing, requires a large phase margin.

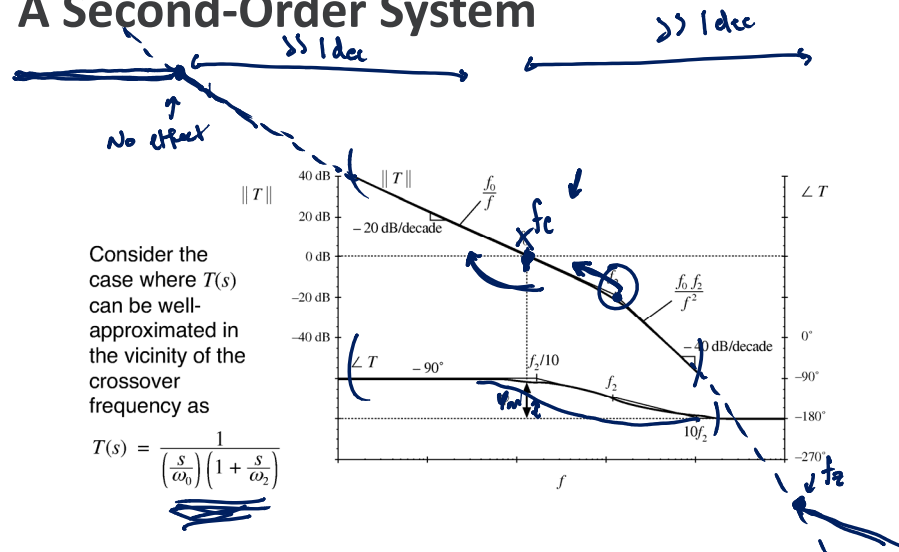
The relation between phase margin and closed-loop  $Q$  is quantified in this section.

## Step Response of Second-Order System

$$T(s) = \frac{1000}{\left(\frac{s}{\omega_1}\right)\left(\frac{s}{\omega_2} + 1\right)}$$



## A Second-Order System



## Closed-Loop Response

If

$$T(s) = \frac{1}{\left(\frac{s}{\omega_0}\right)\left(1 + \frac{s}{\omega_2}\right)}$$

Then

$$\frac{T(s)}{1 + T(s)} = \frac{1}{1 + \frac{1}{T(s)}} = \frac{1}{1 + \frac{s}{\omega_0} + \frac{s^2}{\omega_0 \omega_2}}$$

or,

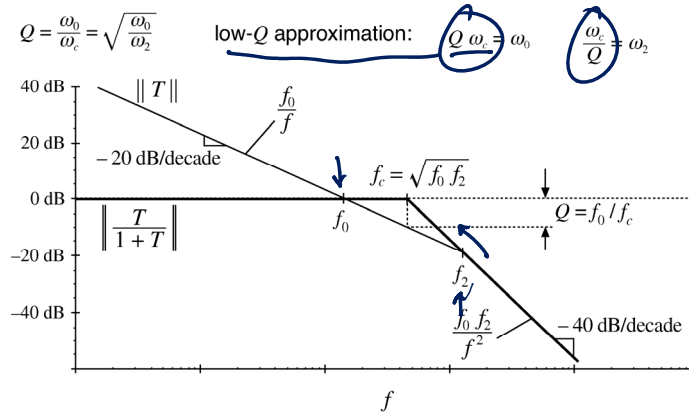
$$\frac{T(s)}{1 + T(s)} = \frac{1}{1 + \frac{s}{Q\omega_c} + \left(\frac{s}{\omega_c}\right)^2}$$

where

$$\omega_c = \sqrt{\omega_0 \omega_2} = 2\pi f_c$$

$$Q = \frac{\omega_0}{\omega_c} = \sqrt{\frac{\omega_0}{\omega_2}}$$

## Low-Q Case

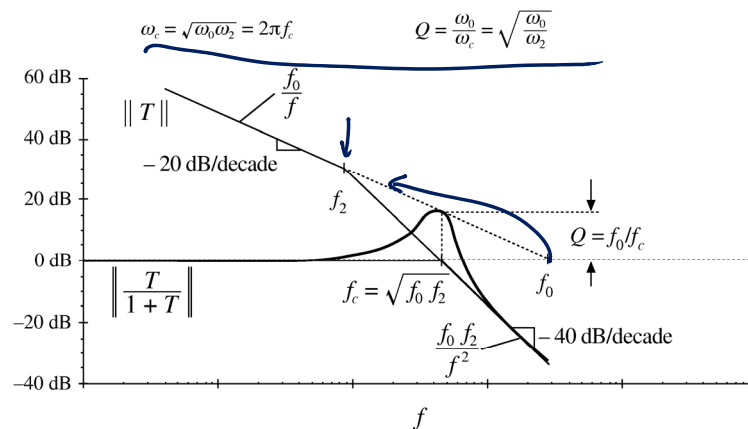


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## High-Q Case



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## $Q$ vs. $\varphi_m$

Solve for exact crossover frequency, evaluate phase margin, express as function of  $\varphi_m$ . Result is:

$$Q = \frac{\sqrt{\cos \varphi_m}}{\sin \varphi_m}$$

$$\varphi_m = \tan^{-1} \sqrt{\frac{1 + \sqrt{1 + 4Q^4}}{2Q^4}}$$

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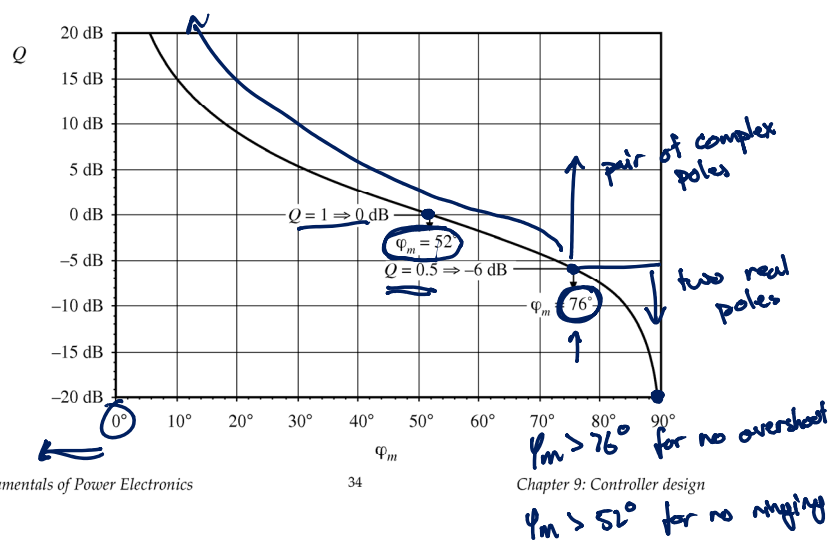
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## $Q$ vs. $\varphi_m$

$Q_{CL} \rightarrow Q$  of Closed-loop Tf



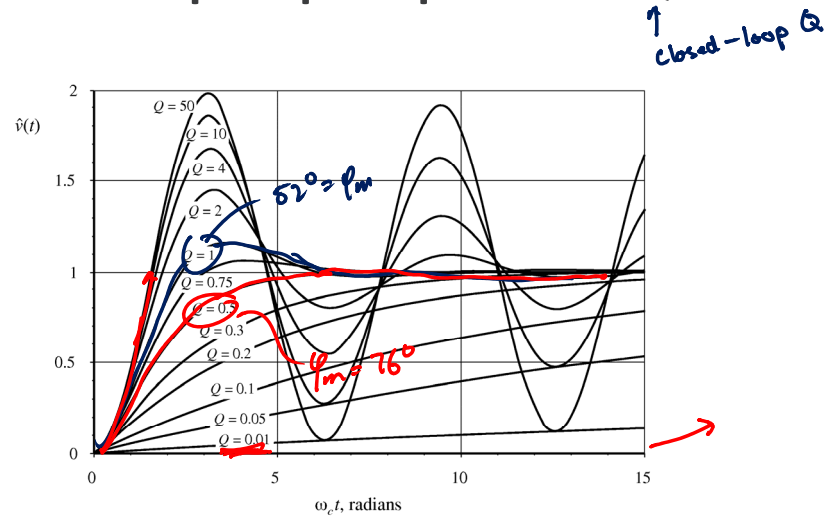
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## Closed-Loop Step Response vs. $Q$



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## 9.5 – Compensator Design

Typical specifications:

- Effect of load current variations on output voltage regulation  
This is a limit on the maximum allowable output impedance
- Effect of input voltage variations on the output voltage regulation  
This limits the maximum allowable line-to-output transfer function
- Transient response time  
This requires a sufficiently high crossover frequency
- Overshoot and ringing  
An adequate phase margin must be obtained

The regulator design problem: add compensator network  $G_c(s)$  to modify  $T(s)$  such that all specifications are met.

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## Design Approach

- Assume  $G_c(s) = 1$ , and plot the resulting uncompensated loop gain  $T_u(s)$
- Examine uncompensated loop gain to determine the needs of the compensator
  - Is low-frequency gain amplitude large enough to result in low steady-state error?
  - Is  $\phi_m$  sufficient for stability and requirements on ringing/overshoot?
  - Is  $f_c$  high enough for a sufficiently fast response?
- Construct compensator to address shortcomings of  $T_u(s)$ 
  - Use “toolbox” of compensators on following slides

$T(s)$   
 w/ our designed  
 $\rightarrow G_c(s)$